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Seat No.

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S.E. (Chemical) (II Semester) EXAMINATION, 2019

PRINCIPLES OF DESIGN

(2015 PATTERN)

Time : Two Hours

Maximum Marks : 50

N.B. :—

(i) Neat diagrams must be drawn wherever necessary.

(ii) Figures to the right indicate full marks.

(iii) Assume suitable data, if necessary.

(iv) Use of calculator is allowed.

1. (a) Derive a relation for the shear stress developed in a shaft when it is subjected to torsion. [6]

(b) A steel rod of 30 mm diameter, 280 mm long is subjected to axial forces alternating between maximum compression of 15 kN and a maximum tension of 5 kN. Find the difference between the greatest and least lengths of the rod, $E = 210 \text{ GPa}$. [6]

Or

2. (a) Distinguish between codes and standards state the engineering aspects covered in standard specifications. [6]

P.T.O.

(b) Horizontal beam ABE is hinged at A and supported on rollers at B. Span $AB = 5 \text{ m}$ and $BE = 2 \text{ m}$. It carries point loads of 2400 N, 3600 N and 1400 N at C, D and E respectively. $AC = 1.5 \text{ m}$, $CD = 2 \text{ m}$, $DB = 1.5 \text{ m}$. All the point loads act vertically. Calculate support reactions and draw S.F. and B.M. diagrams. [6]

3. (a) A shaft rotating at a constant speed is subjected to variable load. The bearing support the shaft are subjected to stationary equivalent radial load of 3 kN for 10% of time, 0.2 kN for 20% of time, 1 kN for 30% of time and no load for remaining time of cycle. If the total life expected for the bearing is 20×10^6 revolutions of 95% reliability, calculate dynamic load rating of the ball bearing. [7]

(b) Discuss the function of coupling. Give at least three practical application. [6]

Or

4. (a) How are ends of belts joined? For horizontal belts which side (tight or slack) of the belt should run on the top and why? [6]

(b) A shaft 80 mm diameter transmits power at maximum shear stress of 63 MPa. Find the length of a 20 mm wide key required to mount a pulley on the shaft so that the stress in the key does not exceed 42 MPa. [7]

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5. (a) What are various types of welding joints used in pressure vessels ? Discuss with neat sketch. [6]
- (b) A thick walled high pressure vessel has 500 mm inside diameter. It is subjected to an internal pressure of 6000 bar, the yield strength of material is 5000 kg/cm². Ultimate tensile strength of material is 6500 kg/cm². Calculate the thickness of vessel according to the various theories of failure. Factor of safety is 1.4.

Also estimate the tangential stress and radial stress variation along the vessel wall. [6]

6. (a) Explain with a neat sketch various types of flanged joints used in pressure vessels. [6]

- (b) A vessel having 1.6 m outside diameter is to operate at a pressure of 5 kg/cm². The permissible stress of the material used for fabrication 1020 kg/cm². Welded joint efficiency is 85%. Calculate the thickness required for a cylindrical vessel and a spherical vessel ? Which vessel should be selected for operation. Design pressure = $5 \times 1.1 = 5.5$ kg/cm² ? [6]

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P.T.O.

7. (a) The inside diameter of cylinder is 25 cm and is subjected to an internal pressure of 600 kg/cm². Allowable tensile stress of the material is 1400 kg/cm². What should be the minimum thickness of the vessel ? [7]
- (b) With neat sketch explain stresses induced in thick vessel subjected to internal pressure. [6]

Or

8. (a) Write a short note on Autofretting of monoblock Pressure vessels. [6]
- (b) A cylindrical pressure vessel 1.8 m in diameter and 5 m in height is subjected to an internal pressure of 8 kg/cm². Corrosion allowance is 2 mm. If the vessel is fabricated as : Class B vessel with $J = 0.85$. [7]

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